

## Chapter 15 - Solved Problems

**Solved Problem 15.1.** *Contributed by - Alvaro Liendo, Universidad Tecnica Federico Santa Maria, Chile.*

*Consider a plant having a nominal model given by*

$$G_o(s) = \frac{1}{s+2} \quad (1)$$

*The aim of the control design is to follow a constant reference. Also, an input disturbance has significant energy in the frequency band  $[0, 6]$  [rad/s]. Find a proper controller using the affine parameterization.*

Solutions to Solved Problem 15.1

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**Solved Problem 15.2.** *Contributed by - Alvaro Liendo, Universidad Tecnica Federico Santa Maria, Chile.*

*Consider a plant having nominal model given by*

$$G_o(s) = \frac{s-2}{(s+1)(s+3)} \quad (2)$$

*Assume that the bandwidth of the closed loop has to be limited to  $8$  [rad/s] in order to have good noise immunity. We further assume that the reference is constant. Design a controller using the affine parameterization.*

Solutions to Solved Problem 15.2

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**Solved Problem 15.3.** *Contributed by - Alvaro Liendo, Universidad Tecnica Federico Santa Maria, Chile.*

*Consider a plant having nominal model given by*

$$G_o(s) = \frac{1}{s^2 + 1.4s + 1} \quad (3)$$

*Find a PID controller that stabilizes the plant and provides zero steady state error at zero frequency.*

Solutions to Solved Problem 15.3

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**Solved Problem 15.4.** *Contributed by - Alvaro Liendo, Universidad Tecnica Federico Santa Maria, Chile.*

*Consider a plant having nominal model given by*

$$G_o(s) = \frac{e^{-s}}{(s+1)(s+2)} \quad (4)$$

*using the Smith controller of Figure 15.5 of the book, find a controller that stabilizes the plant and allows good tracking (although delayed!!) of a reference with energy in the band  $[0, 2]$  [rad/s].*

Solutions to Solved Problem 15.4

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**Solved Problem 15.5.** *Contributed by - Alvaro Liendo, Universidad Tecnica Federico Santa Maria, Chile.*

*Consider a plant having nominal model given by*

$$G_o(s) = \frac{1}{s-1} \quad (5)$$

*Find a controller that stabilizes the plant, ensures perfect reference tracking at zero frequency and provides good compensation of disturbances with energy in the frequency band  $[0, 1]$  [rad/s].*

Solutions to Solved Problem 15.5

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**Solved Problem 15.6.** *Contributed by - Alvaro Liendo, Universidad Tecnica Federico Santa Maria, Chile.*

*Consider a plant having nominal model given by*

$$G_o(s) = \frac{1}{s-1} \quad (6)$$

*Find all stabilizing controllers for this plant.*

Solutions to Solved Problem 15.6

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**Solved Problem 15.7.** *Contributed by - Alvaro Liendo, Universidad Tecnica Federico Santa Maria, Chile.*

*Show that the controller found in Problem 15.5 is consistent with the solution to Problem 15.6*

Solutions to Solved Problem 15.7

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**Solved Problem 15.8.** *Contributed by - Alvaro Liendo, Universidad Tecnica Federico Santa Maria, Chile.*

*Consider a discrete time plant having nominal transfer function given by*

$$G_{oq}(z) = \frac{z+2}{(z-0.9)(z-0.1)} \quad (7)$$

*Find a controller that stabilizes the plant, ensure perfect tracking of a constant reference and exhibits a closed-loop step response faster than  $(0.7)^k$ .*

Solutions to Solved Problem 15.8

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**Solved Problem 15.9.** *Contributed by - Alvaro Liendo, Universidad Tecnica Federico Santa Maria, Chile.*

*Consider a discrete time plant having nominal transfer function given by*

$$G_{oq}(z) = \frac{1}{(z+2)} \quad (8)$$

*Find all controllers that stabilize the plant and ensure zero steady state error when tracking a constant reference.*

Solutions to Solved Problem 15.9

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## Chapter 15 - Solutions to Solved Problems

**Solution 15.1.** To achieve perfect tracking,  $T_o(0) = 1$  is needed.

Also, we have that

$$S_{io}(s) = [1 - T_o(s)]G_o(s) = \frac{1}{(s+2)}[1 - T_o(s)] \quad (9)$$

Then, to achieve good disturbance rejection, we need the bandwidth of  $T_o(s)$  to be at least  $6[\text{rad/s}]$ . Also  $[1 - T_o(s)]$  should cancel the factor  $(s+2)$  in  $S_{io}$ . This leads to  $T_o(-2) = 1$ . We choose

$$Q(s) = \frac{36(\tau s + 1)(s + 2)}{s^2 + 8s + 36} \quad (10)$$

The condition  $T_o(-2) = Q(-2)G_o(-2) = 1$  implies  $\tau = \frac{1}{6}$ . Thus

$$Q(s) = \frac{6(s+6)(s+2)}{s^2 + 8s + 36} \quad (11)$$

The quality of the disturbance compensation can be evaluated by plotting the magnitude of the input sensitivity function. This is shown in Figure 1.

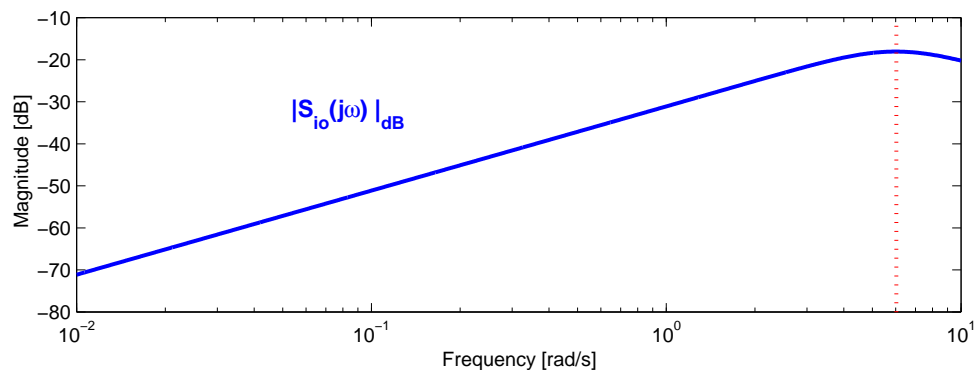


Figure 1: Input sensitivity frequency response

From Figure 1 we see that, in the disturbance frequency band,  $[0, 6] [\text{rad/s}]$ , the gain is at most  $-17 [\text{dB}]$ .

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**Solution 15.2.** We know that  $(s-2)$  must be a zero of  $T_o(s)$  to ensure an internally stable closed loop. Also,  $T_o(0)$  must be zero to ensure perfect steady state reference tracking. Also, to achieve the fastest possible loop, we set the poles in  $T_o(s)$  close to  $8[\text{rad/s}]$ . Say we choose

$$T_o(s) = \frac{-32(s-2)}{(s^2 + 12s + 64)} \quad (12)$$

This leads to

$$Q(s) = \frac{-32(s+1)(s+3)}{(s^2 + 12s + 64)} \quad (13)$$

**Solution 15.3.** We use the theory developed in subsection §15.4.3 of the book. To achieve a proper inverse we select

$$Q(s) = F_Q(s)[G_o(s)]^{-1} = F_Q(s)(s^2 + 1.4s + 1) \quad (14)$$

We choose a natural frequency of 2.5 and a damping factor of 0.65. Hence

$$F_Q(s) = \frac{1}{0.16s^2 + 0.52s + 1} \quad (15)$$

And we then have

$$Q(s) = \frac{s^2 + 1.4s + 1}{0.16s^2 + 0.52s + 1} \quad (16)$$

Using equations (15.4.16) to (15.4.19) in the book we finally have

$$K_P = 2.10 \quad (17)$$

$$K_I = 1.92 \quad (18)$$

$$K_D = 0.13 \quad (19)$$

$$\tau_D = 0.31 \quad (20)$$

The closed-loop step response using this controller is shown in Figure 2.

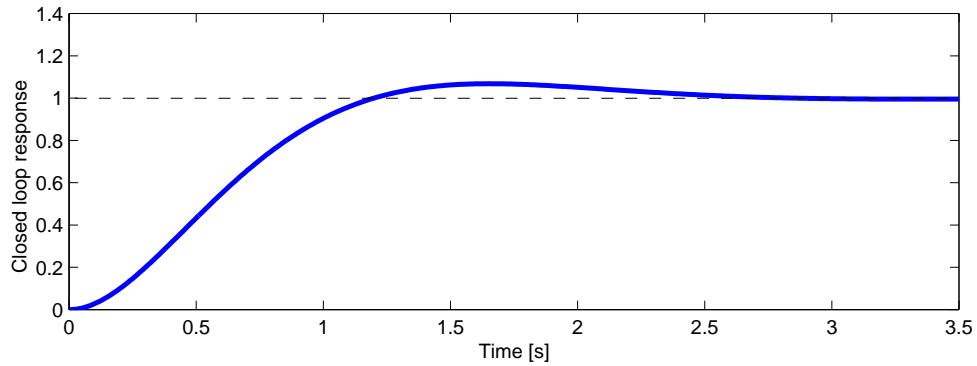


Figure 2: Closed-loop step response

**Solution 15.4.** Using the structure of Figure 15.5 in the book, we find a controller for the rational part of  $G_o(s)$ , denoted by  $\overline{G}_o(s)$ . An appropriate  $Q(s)$  (achieving a closed-loop bandwidth larger than 2 [rad/s]) is

$$Q(s) = \frac{20(s+1)(s+2)}{(s^2 + 2.8s + 4)(s+10)} \quad (21)$$

leading to

$$T_o(s) = 40 \frac{e^{-s}}{(s^2 + 2.8s + 4)(s+10)} \quad (22)$$

Note that  $T_o(0) = 1$ , i.e., the controller provides integral action.

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**Solution 15.5.** To ensure internal stability of the closed loop it is necessary and sufficient that  $Q(s)$  be stable and that  $(s-1)$  be a zero of  $Q(s)$  and of  $(1 - G_o(s)Q(s))$ . We choose

$$Q(s) = (s-1)\overline{Q}(s) \quad (23)$$

where

$$\overline{Q}(s) = \frac{P(s)}{(s+2)(s+3)} \quad (24)$$

with  $P(s)$  a polynomial satisfying  $P(0) = 6$ , to achieve zero steady state error for constant references. (This forces  $\overline{Q}(0) = 1$ .)

Furthermore,

$$1 - G_o(s)Q(s) = 1 - \overline{Q}(s) = \frac{s^2 + 5s + 6 - P(s)}{(s+2)(s+3)} \quad (25)$$

must have a zero at  $s = 1$  and one other zero. This leads to

$$s^2 + 5s + 6 - P(s) = (s-1)(s+\alpha) \quad (26)$$

$$P(s) = (6-\alpha)s + (6+\alpha) \quad (27)$$

To satisfy  $P(0) = 6$ , we choose  $\alpha = 0$ . This leads to

$$\overline{Q}(s) = \frac{6s+6}{(s+2)(s+3)} \quad (28)$$

$$Q(s) = \frac{6(s+1)(s-1)}{(s+2)(s+3)} \quad (29)$$

With this choice we have that

$$T_o(s) = Q(s)G_o(s) = \frac{6(s+1)}{(s+2)(s+3)} \quad (30)$$

Note that  $T_o(s)$  satisfies the interpolation constraint  $T_o(1) = 1$ .

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**Solution 15.6.** We know that all controllers that stabilize this plant have the form (see equation (15.7.11) in the book)

$$Q(s) = \frac{s-1}{F(s)} \left[ \frac{P(s)}{E(s)} + Q_u(s) \frac{(s-1)}{E(s)} \right] \quad (31)$$

where  $Q_u(s)$  is any stable transfer function;  $F(s)$ ,  $E(s)$  are stable polynomials and  $P(s)$ ,  $F(s)$ ,  $E(s)$  satisfy the pole assignment equation

$$(s-1)L(s) + P(s) = E(s)F(s) \quad (32)$$

$$sL(s) + (P(s) - L(s)) = E(s)F(s) \quad (33)$$

Note that  $P(s)/L(s)$  is any stabilizing controller for this plant. Choosing  $L(s) = 1$ ,  $P(s) = 2$  and  $E(s) = 1$  we have  $F(s) = s+1$ . This leads to

$$Q(s) = \frac{(s-1)}{(s+1)} [2 + (s-1)Q_u(s)] \quad (34)$$

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**Solution 15.7.** We only need to find the  $Q_u(s)$  in Problem 15.6 that leads to  $Q(s)$  in Problem 15.5 and verify that is stable

$$\frac{(s-1)}{(s+1)} [2 + (s-1)Q_u(s)] = \frac{6(s+1)(s-1)}{(s+2)(s+3)} \quad (35)$$

After some algebra, we find

$$Q_u(s) = \frac{4s+6}{(s+2)(s+3)} \quad (36)$$

This shows consistency because  $Q_u(s)$  is found to be stable as required in the solution to Problem 15.6.

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**Solution 15.8.** To have a stable  $Q_q(z)$ ,  $T_{oq}(z)$  must contain the unstable zero of  $G_{oq}(z)$ . Also, to follow a constant reference we need  $T_{oq}(1) = 1$ . Finally, to ensure that the closed-loop step response has transients faster than  $(0.7)^k$ ,  $T_{oq}(z)$  must contain only poles inside the disk with radius 0.7. This requires that  $Q_q(z)$  cancels the slow plant pole (located at  $z = 0.9$ ). Considering this, we choose

$$T_{oq}(z) = K \frac{z+2}{(z-0.1)^2} \quad (37)$$

The condition  $T_{oq}(1) = 1$  leads to  $K = 0.27$  and

$$T_{oq}(z) = 0.27 \frac{z+2}{(z-0.1)^2} \quad (38)$$

This yields

$$Q_q(z) = 0.27 \frac{(z-0.9)}{(z-0.1)} \quad (39)$$

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**Solution 15.9.** We first find all stabilizing controllers for this plant. They have the form (see equation (15.7.11) in the book)

$$Q_q(z) = \frac{z+2}{F_q(z)} \left[ \frac{P_q(z)}{E_q(z)} + Q_{uq}(z) \frac{(z+2)}{E_q(z)} \right] \quad (40)$$

where  $Q_{uq}(z)$  is any stable transfer function;  $F_q(z)$ ,  $E_q(z)$  are stable polynomials and  $P_q(z)$ ,  $F_q(z)$ ,  $E_q(z)$  satisfy the pole assignment equation

$$(z+2)L_q(z) + P_q(z) = E_q(z)F_q(z) \quad (41)$$

$$zL_q(z) + [P_q(z) + 2L_q(z)] = E_q(z)F_q(z) \quad (42)$$

Note that  $P_q(z)/L_q(z)$  is any stabilizing controller for this plant. Choosing  $L_q(z) = 1$ ,  $P_q(z) = -2.5$  and  $E_q(z) = 1$  we have  $F_q(z) = z - 0.5$ . This leads to

$$Q_q(z) = \frac{(z+2)}{(z-0.5)} [-2.5 + (z+2)Q_{uq}(z)] \quad (43)$$

This parameterizes all controllers that stabilize the plant. We next need to ensure perfect reference tracking. This is achieved by setting  $Q_q(1) = [G_{oq}(z)]^{-1} = 3$ . Hence, we require

$$Q_q(1) = 6[-2.5 + 3Q_{uq}(1)] = 3 \quad (44)$$

This leads to  $Q_{uq}(1) = 1$ . The only other requirement is that  $Q_{uq}(z)$  must be stable.